BITS, PILANI – DUBAI DUBAI INTERNATIONAL ACADEMIC CITY, DUBAI

Il Year Il Semester : 2008 - 2009 Course: ME C 211 Applied Thermodynamics

Comprehensive Examination [Closed Book]

Max.Marks: 80 Weightage: 40 %

Section VI - Mechanical

Date: 20-05-2009 Time: 3 hours

Note: (i) Answer all Question in a sequence (ii) Assume suitable value if required and clearly state them (iii) Thermodynamics tables are permitted (iv) Answer in the BLUE COLOUR main answer book only

- Q.1. For an engine working on the ideal Dual cycle, the compression ratio is 10 and the maximum pressure is limited to 70 bar. If the heat supplied is 1680 kJ /kg, find the pressures and temperatures at the various salient points of the cycle and the cycle efficiency. The pressure and temperature of air at the commencement of compression are 1 bar and 100 °C respectively. Assume C_p = 1.004 kJ /kg-K and C_v = 0.717 kJ /kg-K for air.
- Q.2. A one-litre cubic capacity, four-cylinder, four-stroke SI engine has a compression ratio of 8 and bore of 100 mm, with stroke equal to the bore. The volumetric efficiency of each cylinder is equal to 75 %. The engine operates at a speed of 4800 rpm with an air-fuel ratio 15. Given that the calorific value of fuel = 42 MJ / kg, atmospheric density = 1.12 kg / m³, mean effective pressure in the cylinder = 10 bar and mechanical efficiency = 80 %, determine the clearance volume of the engine, indicated thermal efficiency and brake power.
- Q.3. In a steam power cycle, the steam supply is at 15 bar and dry and saturated (X = 100%) is expanded to 0.4 bar. Assume ideal processes, Calculate the Rankine efficiency of the cycle. Neglect pump work.
- Q.4. A single-cylinder double-acting steam engine runs at 250 rpm and indicated power is 30 kW with cut-off at 40 % of stroke. The pressure limits of operation are 10 bar and 1 bar. The L / D ratio is 1.25 and the diagram factor is 0.75. Assume dry saturated steam at inlet hyperbolic expansion and negligible effect of piston rod. Find (i) mean effective pressure(ii)cylinder dimensions and (iii)indicated thermal efficiency.[10 Marks]

- Q.5. The LDO oil is to be cooled by water in a surface condenser. The oil flows steadily through a bundle of metal tubes submerged 90 °C and leaves at 30 °C. The cooling water enters at 25 °C and leave at 70 °C. The enthalpy of oil at t °C is given by h = 1.68 t + 10.5 X 10⁻⁴ t² kJ / kg. What is the cooling water flow required for cooling 2.78 kg/s of oil? Neglect the sp.heat for oil. Take sp heat of H₂O= 4.187 kJ / kg-K [8 Marks]
- Q.6. The sling psychrometer in a laboratory test recorded the following readings; Dry bulb temperature = 35 °C, wet bulb temperature = 25 °C, Take atmospheric pressure = 1.0132 bar. Calculate (i) specific humidity (ii) relative humidity (iii) vapour density in air (iv) dew point temperature (v) enthalpy of mixture per kg of dry air. [6 Marks]
- Q.7. In an air-standard regenerative gas turbine cycle the pressure ratio is 5. Air enters the compressor at 1 bar, 300K and leaves at 490K. The maximum temperature in the cycle is 1000K. Calculate the cycle efficiency, given that the efficiency of the regenerator and the adiabatic efficiency of the turbine are each 80 %. Assume for air, the ratio of specific heats is 1.4. Also show the cycle in T-S diagram. [10 Marks]
- Q.8. F-12 vapour compression refrigeration cycle operating between an evaporator temperature of -10°C and a condenser temperature of 40°C. The compressor employed is of 20 cm diameter and 15 cm stroke, twin cylinder, single acting compressor having a volumetric efficiency of 85 % and runs at 500 RPM. Take enthalpy at the end of compression is 220 kJ / kg, the ambient temperature is 37 °C, specific volume of the refrigerant at the compressor inlet 0.08 m³ / kg. Calculate:

 (a) the amount of refrigerant circulated per minute (b) cooling capacity in TR

 (c) compressor power (d) COP of the system (e) carnot COP and (f) relative COP. Assume 1 TR = 14000 kJ / hr. Use the properties of F-12 as listed below: [10 Marks]

T (°C)	P(Mpa)	h _f (kJ/kg)	h _g (kJ/kg)	
- 10	0.2191	26.85	183.1	
40	0.9607	74.53	203.1	

Q.9. [a] Explain the effect of area change in subsonic and supersonic flows with sketch[5M]
Q.10. [b] Draw the single stage reciprocating compressor on p-V diagram and indicate various salient points of the cycle.
[4 Marks]

o my ling of dry oir-

0.006

0.008

0.010

0.012

Specific humidity, kg/kg of dry air

8,0.0

0,022

0.024

BITS, Pilani –Dubai

Dubai International Academic City, Dubai, U.A.E II Year II Semester 2008-2009 [Mechanical]

Test No.2 (Open Book)

Course No. ME C 211

Course Title: Applied Thermodynamics Weightage: 20 %

Date: 30 -04-2009

Max.Marks: 20

Duration: 50 min.

Notes:

Answer all the questions

Draw neat sketches wherever necessary

Thermodynamics Tables are permitted.

Make suitable assumptions if required and clearly state them

An ammonia refrigerator produces 20 tons of ice per day from and at 0°C. The condensation and evaporation takes at 20°C and - 20°C respectively. The temperature of vapour at the end of isentropic compression is 50 °C and there is no under cooling of liquid. The actual COP is 70 % of the theoretical COP. Use the properties of NH3 as listed below:

Temperature	Entha	lpy (KJ / kg)	Entropy (KJ / kg-K)				
°C	h _f	h _g	Sf	Sg			
20	274.98	1461.58	1.04341	5.0919			
-20	89.72	1419.05	0.3682	5.6204			

Take: (i) The relation of enthalpy for super heated vapour is $h = h_q + c_p (T_2 - T_1)$ (ii) Specific heat for superheated vapour = 2.8 kJ/ Kg-k, (iii) specific volume of the dry vapour of the refrigerant at the compressor inlet (v) = 0.624 m³ / kg, (vi) The fusion of ice = 335 kJ / Kg.

Determine: [a] The refrigerating capacity of the plant [b] Power required to drive the compressor [c] The mass flow rate of NH3circulation [d] The cylinder dimensions of the single-acting compressor if the speed is 240 RPM, assuming L = D and volumetric efficiency is 80 %. [7 Marks]

- Q.2. A single-cylinder double-acting steam engine with 15 cm bore and 20 cm stroke is developed 20 kW at 300 rpm with cut-off occurring at 20 % of stroke. Back pressure is of the steam is 0.28 bar. Determine admission of the steam if diagram factor is 0.72. Also calculate mass flow rate of steam and indicated thermal efficiency. Assume dry saturated steam at inlet hyperbolic expansion and negligible of area of the piston rod. [7 Marks]
- Q.3. Refrigerant-134 a is to be cooled by water in a surface condenser. The refrigerant enters the condenser with mass flow rate of 6 kg / min at 1 MPa and 70 °C and leaves at 35 °C. The cooling water enters at 15 °C and leave at 25 °C. Neglect the specific heat of both fluid.

Calculate:

(i)) Mass	flow rate	of	cooling	water	required
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(ii) Heat transfer rate from the refrigerant to water.

[6 Marks]

BITS, Pilani –Dubai

Dubai International Academic City, Dubai, U.A.E. Il Year Il Semester 2008-2009 [Mechanical]

Test No.1 (Closed Book)

Course No. ME C 211

Course Title: Applied Thermodynamics Weightage: 25%

Date: 22-03-2009

Max.Marks: 25

Duration: 50 min.

Notes:

- Answer all the questions
- Draw neat sketches wherever necessary
- Thermodynamics Tables are permitted.
- Make suitable assumptions if required and clearly state them
- Q.1. What is meant by crankcase scavenged two stroke engine?

[3M]

- Q.2. A single- cylinder, four-stroke hydrogen fuelled spark-ignition engine delivers a brake power of 20 kW at 6000 rpm. The air-gas ratio is 8:1 and the calorific value of fuel is 11000 kJ /m³. If volumetric efficiency is 70 %, Indicated efficiency is 33 %, and the mechanical efficiency is 90 %, calculate the cubic capacity of the engine. [7M]
- Q.3. Draw the p-V diagram of a Dual cycle and mark the various processes. Why this cycle is also called limited pressure or mixed cycle? [4M]
- Q.4. Determine the ideal efficiency of the diesel engine having a cylinder with bore 250 mm, stroke 375 mm and clearance volume of 1500 cc, with fuel cut-off occurring at 5 % of the stroke. Assume y = 1.4 for air. [6M]
- A Steam turbine receives steam at a pressure of 1 MPa and temperature of 300 °C. The Q.5. saturated steam leaves the turbine at a pressure of 15 kPa. The actual work output of the turbine is measured and is found to be 600 KJ / kg of steam flowing through the turbine. Determine the isentropic efficiency of the turbine. [5M]

Subject: Applied Thermodynamics Duration: 15 Min							Cour	se No: M DATE: Max		/ 09
Name of the student:QUIZ III- QUESTION							I.D.:	* * *		
aQ	1	2	3	4	5	6	7	8	9	10
Ans										
_ `	-			g steam er 3 and back	_	-		•		
effective	e pressur	e.								
[A] 700	kN/m ²	[B]	695 kN /	' m ² [C]	699 l	$\kappa N / m^2$		[D] 697	7 kN / m	2
2. In a engine c		ngine D-	slide va	lve control	s the	flow of st	eam ir	nto and o	ut of the	e steam
[A] Thro	ough one	e admiss	ion port	and two e	xhaust	ports [E	3] Thro	ough one	admissi	on port
and one	exhaust	ports [C	Through	gh one exh	aust p	ort and tv	vo adm	ission po	rts	
[D] Thro	ough two	exhaust	port an	d two adm	ission	ports		·		
	•			ng steam e	•			ŕ	,	
				ate mass fl		ie oi steai	III. ASS	ume spec	onic von	ume or
				s 0.194 m ³	_					
[A] 0.16	3 kg/s		[B]	0.12 3 kg/	s [C	CJ 0.14 3 1	kg/s	[D] 0.1	5 3 kg/s	
4. The C	OP for a	a carnot i	efrigera	tor is		than t	hat of	carnot he	at pump).
[A] Less	[B]	More	Г	C] One-th	ird les	SS	[D] O	ne-third 1	nore	

5. A carnot refrig	gerator rec	juires 1.3 kW	per tonne of refrig	eration to	maintain a region at
low temperature	of – 38 °C	. Calculate the	maximum tempe	rature of th	ne cycle. Assume on
tonne of refrigera	ition is 140	000kJ/h	-		•
[A] 43 °C			[C] 42 °C	[D] 40 °C	C
6. Refrigerating	system op	erates on the r	eversed carnot cyc	cle. The hig	gher temperature of
the refrigerant in	the systen	n is 35 °C and	the lower tempera	ture is - 15	5 °C, the capacity is
to be 12 tonnes.	Assume on	tonne of refri	geration is 14000	kJ/h. Find	work input.
[A] 33558 kJ/h	[B]	34558 kJ/h	[C] 32558	kJ/h	[D] 31558 kJ/h
7. In steam powe	r plant, the	e function of a	condensor is		
[A] To maintain j	pressure b	elow atmosphe	eric to increase wo	ork output	from the prime
mover [B] To red	eive large	volume of ste	am exhausted from	n steam pr	ime mover
[C] To condense	large volu	me of steam to	water which may	y be used a	gain in boiler
[D] All of the abo	ove				
8. In Jet type cor	ndensers				
[A] Cooling water	r passes th	nrough tubes a	nd steam surround	ls them	
[B] Steam passes	through to	ubes and cooli	ng water surround	ls them [C]	Cooling water and
steam mix [D] co	oling wate	er and steam de	o not mix		
9. The outlet and	inlet temp	eratures of co	oling water to a co	ondenser ar	re 37.5 °C and
30 °C respectivel	y. The satu	uration steam t	emperature is 40 °	C. Find ou	at the condenser
efficiency.					
[A] 75.3 %	[B] '	75.1 %	[C] 75.2 %	I]	O] 75 %
10. In a surface c	ondenser i	f air is remove	ed, there is		
[A] fall in absolu	te pressure	e maintained ir	n condenser [B] R	ise in abso	lute pressure
maintained in cor	ndenser [C] No change in	absolute pressure	e maintain	ed in condenser
[D] Rise in temper	rature of	condensed stea	ım		
		*****	******	******	**a

Subject: Applied Thermodynamics							Cours	se No: M DATE:			
Duratio	n: 15 M	in	(م الروة	W 015				Marks:		
Name o	f the stu	dent:					I.D.: -		********		
				2	OUIZ II						
bQ	1	2	3	4	5	6	6 7 8 9				
Ans											
1. In a	steam er	ngine D-	slide val	ve contr	ols the f	flow of s	team in	to and or	it of the	steam	
engine c	ylinder										
[A] Thro	ough one	admiss	ion port	and two	exhaust	ports []	B] Thro	ugh one	admissio	n port	
and one	exhaust	ports [C] Throug	gh one ex	khaust po	ort and to	wo admi	ssion po	rts		
[D] Thro	ough two	exhaust	port and	d two ad	mission	ports					
2. The C	OP for a	a carnot i	efrigera	tor is		than	that of o	carnot he	at pump		
[A] Less	[B]	More	[C] One-	third les	ss	[D] O	ne-third r	nore		
3. Refrig	gerating	system o	perates	on the re	eversed o	arnot cy	cle. The	higher t	emperati	ure of	
the refrig	gerant in	the syste	em is 35	°C and t	the lower	r tempera	ature is	- 15 °C, t	he capac	ity is	
to be 12	tonnes.	Assume	on tonne	of refrig	geration	is 14000	kJ/h. Fi	nd work	input.		
[A] 3355	88 kJ/h	[]	34558	8 kJ/h	[C	32558	kJ/h	[D]	315581	cJ/h	
4. In Jet	type con	ndensers									
[A] Cool	ing wate	er passes	through	tubes ar	nd steam	surroun	ds them				
[B] Stear	n passes	through	tubes ar	nd coolin	ng water	surround	ds them	[C] Cool	ing wate	er and	
steam mi	x [D] co	oling wa	iter and	steam do	not mix	:					
5. In a su	rface co	ndenser	if air is r	emoved,	, there is						
[A] fall is	n absolu	te pressu	ıre maint	ained in	condens	ser [B] R	ise in at	solute p	essure		
maintain	ed in cor	ndenser [C] No c	hange in	absolute	e press ur	e mainta	ained in o	ondense	er	
[D] Rise	in tempe	erature o	f conden	sed stear	m						

MN/m ² , compression effective pressure.	on ratio is	3.33 and back	x pressure is 0.13	boiler inlet pressure i MN / m ² . Find the m	ean
[A] 700 kN / m ²	[B] 695 k	kN/m² [C	[] 699 kN / m ²	[D] 697 kN / s	m²
7. A single-cylinde	r double-a	cting steam	engine with 15	em bore, 20 cm stroke	e, at 300
rpm with cut-off ra	atio 5. Cal	culate mass f	low rate of steam	m. Assume specific vo	olume of
the dry saturated s	team at inl	et is 0.194 m	³ /kg.		
[A] 0.16 3 kg/s		[B] 0.12 3 kg	/s [C] 0.14 3	kg/s [D] 0.15 3 kg/	's
	– 38 °C. C	alculate the r		eration to maintain a retature of the cycle. Ass	_
[A] 43 °C [E	3] 44 °C	[0	C] 42 °C	[D] 40 °C	
mover [B] To receive	essure belo ve large vo ge volume	w atmospher	te to increase we	ork output from the print or steam prime mover or be used again in boile	
	The saturat	tion steam ter 1 %	_		

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Subject: Applied Thermodynamics								Course No: ME C 211 DATE: 25/ 03/09				
Duratio	n: 15 M	L in		_					25/ 03 . Marks:			
Name o	f the sti	rdent:		(Q) (Cs7	<u> </u>	r ID.					
name o	ame of the student:			<u>)</u>	DUIZ II		1.D.; -	·				
cQ	1	2	3	4	5	6	7	8	9	10		
Ans												
[A] Char	nge in in	cle the voternal en	ergy bet	ween inl	et and or	_	given by_		_			
[C] Chan								4				
- -	•	•				ι						
[D] Chan	ge in en	thalpy be	etween 1	nlet and	outlet							
							pression i					
[A] 50.6	% [B] 49.6 %		[C] 48.6 %		[D] 47.6	%				
3. For an clearance [A] 1.9 cc	volume	0.2 cc. F	ind the	Swept V	olume.		pression 1	ratio is 1	0 and			
4. A four power of t									•			

engine. Assume $\gamma = 1.4$.

[A] 65.5 kJ/s [B] 66.5 kJ/s [C] 67.5 kJ/s [D] 68.5 kJ/s

5. In steam power plant power output from the turbine is 1665 kW and pump work is kW. Heat input to the plant is 3000 kW. Find the efficiency of the power plant. [A] 55 % [B] 56 % [C] 54 % [D] 53 %	15
6. The air standard Otto cycle consists of	
[A] Two constant volume and two isentropic processes	
[B] Two constant pressure and two isentropic processes	
[C] Two constant pressure and two constant volume processes	
[D] Two isothermal and two adiabatic process	
7. For given compression ratio the work output of Otto cycle is	
[A] Increase with increase 'r' [B] Decrease with increase 'r' [C] is not affected	
[D] None of the above	
8. For the same compression ratio and heat addition	
$[A] \ \eta_{\ Orto} > \eta_{\ Diesel} > \eta_{\ Dual} \ [B] \ \eta_{\ Diesel} > \eta_{\ Orto} > \eta_{\ Dual} \ [C] \ \eta_{\ Orto} > \eta_{\ Dual} > \eta_{\ Diesel} $	
[D] $\eta_{\text{Dual}} > \eta_{\text{Diesel}} > \eta_{\text{Otto}}$.	
9. The air standard Diesel cycle consists of	
[A] Two constant volume and two isentropic processes	
[B] Two constant pressure and one isentropic processes and one constant volume	
[C] Two isentropic processes one constant volume and one constant pressure.	
D] Two isothermal and two adiabatic process	
0. Rankine cycle efficiency of a good steam power plant may be in the range of	
A] 15 to 20 % [B] 70 – 80 % [C] 90 – 95 %	
D] 35 – 45 %	

		ct: Applic		modyna —	/1	(10 N	4		DATE:	IE C 212 25/ 02/ . Marks:	09
Na	me	of the stu	dent:		سي ية ل د الاستجد برح	OUIZ I		I.D.: -			
(Q	1	2	3	4	5	6	7	8	9	10
	٠		٠								
1.		Gudgeon [A] Pisto: connectin	n and bi	g end of	connect	ing rod					
2.		The range	e of volu	ımetric e	efficienc	y of dies	el engin	e is			
		[A] 65 –	75 % [B	3] 75 – 8:	5 % [C]	85 – 90	% [D] 9	0 – 95 %	b		
3.	a)	Relative of Actual th		•							
		Mechani	cal effic	eiency							
	b)	Actual the	ermal ef	ficiency							
		Air-Stand	lard effi	ciency							
	c)	Air-standa	ard effic	iency							
		Actual the	ermal ef	ficiency							
	d)	Indicated	thermal	efficien	су						
		Actual the	ermal ef	ficiency	·						
4.	E	Equivalenc	e ratio is	S							
		[A] Actua	l fuel-ai	r ratio / :	Stoichio	metric fu	ıel-air ra	tio			
		[B] Stoich	niometric	c fuel-air	ratio / A	Actual fu	iel-air ra	tio			
		[C] Theor	etical fu	el-air ra	tio / Stoi	chiomet	ric fuel-a	air ratio			
		[D] Stoich	niometri	c fuel-air	ratio / [Theoretic	cal fuel-a	air ratio			

5.	The stroke to bore ratio, where d > L is
	[A] Square engine [B] Medium Square engine [C] Under square- engine [D] Over Square- engine
6.	The mechanical efficiency of a single cylinder 4 stroke engine is 80 %. The
	frictional power is estimated to be 25 kW. Calculate the indicated power.
	[A] 120 kW [B] 130 kW [C] 121 kW [D] 125 kW
7.	A four stroke engine brake thermal efficiency is 24.96 % with running at a fixed
	speed. The fuel consumption rate is 20 kg/h. Find out the brake power.
	Assume $CV = 42000 \text{ kJ} / \text{kg}$.
	[A] 58 kW [B] 60 kW [C] 56 kW [D] 57 Kw
8.	In a 4 cylinder, 4 stroke hydrogen fuelled SI engine, the actual volume of air inducted in to cylinder is 1105.44 cc with volumetric efficiency of 70 %. Calculate the cubic capacity of the engine. [A] 6314.8 cc [B] 6318.8 cc [C] 6320.8 cc [D] 6316.8 cc
9.	A 4 stroke SI engine at full load delivers 50 kW with brake thermal efficiency of 25 %. Find out volume flow rate of fuel. Assume CV of fuel 42000 kJ / kg, and specifi gravity of petrol is 0.75.
	[A] $5.34 \times 10^{-6} \text{ m}^3/\text{s}$ [B] $7.34 \times 10^{-6} \text{ m}^3/\text{s}$ [C] $6.34 \times 10^{-6} \text{ m}^3/\text{s}$ [D] $8.34 \times 10^{-6} \text{ m}^3/\text{s}$
10.	A 4 stroke four cylinder diesel engine running at 2000 rpm. The piston stroke length is 100 mm. Find the piston speed.
	[A] 5.67 m/s [B] 6.00 m/s [C] 6.67 m/s [D] 5.99 m/s
