

**BITS, PILANI-DUBAI, ACADEMIC CITY, DUBAI**  
**FIRST SEMESTER 2010-2011**  
**CHE C351 HEAT TRANSFER OPERATIONS**  
**COMPREHENSIVE EXAMINATION**

DATE: 28-12-10

**DURATION: 3 hours**

**MAXIMUM MARKS: 35**

**Note: Attempt ALL questions. Mention appropriate units in your answers. Without units, the answer will not be deemed as correct, even if the numerical value is correct.**

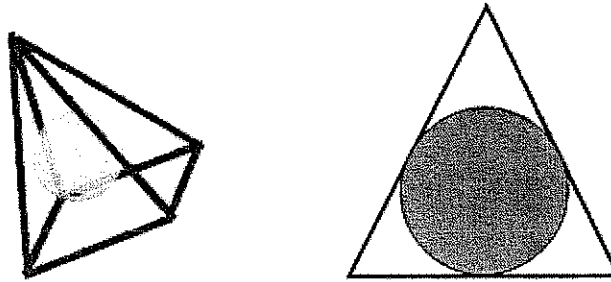
- 1(a) A spherical vessel of 50 cm outside diameter is insulated with 20 cm thickness of insulation ( $k = 0.04 \text{ W/m}^\circ\text{C}$ ). The surface temperature of the vessel is  $-195^\circ\text{C}$  and outside air is at  $10^\circ\text{C}$ . Calculate the heat loss. [2]
- 1 (b) A slab of 12 cm thickness and generating heat uniformly at  $1 \text{ MW/m}^3$  has thermal conductivity =  $200 \text{ W/m}^\circ\text{C}$ . Both surfaces of slab are maintained at  $150^\circ\text{C}$ . Determine the maximum temperature and its location. [2]
- 2 An array of 10 fins ( $k = 180 \text{ W/m}^\circ\text{C}$ ) is used to cool a transistor in an environment with  $h = 12 \text{ W/m}^\circ\text{C}$  and ambient temperature  $350^\circ\text{C}$ . Each fin is 3 mm wide X 0.4 mm thick X 5 mm long and has base temperature  $60^\circ\text{C}$ . Assume fin-tips to be insulated.
- (a) Determine the power dissipated by the fin array.
- (b) What is the efficiency of each fin?
- (c) Now, if the length of each fin is doubled, what will be the power dissipated? [4+1=1=6]
- 3 A 12-mm diameter steel sphere ( $k = 42.5 \text{ W/m}^\circ\text{C}$ ) is exposed to air at  $27^\circ\text{C}$  with  $h = 114 \text{ W/m}^2^\circ\text{C}$ . Determine:
- (a) Time required to cool the sphere from  $540^\circ\text{C}$  to  $95^\circ\text{C}$ ?
- (b) Total heat transferred during this period. [3+1=4]
- Given: density =  $7850 \text{ kg/m}^3$ , heat capacity =  $475 \text{ J/kg}^\circ\text{C}$ ,  
Thermal diffusivity =  $1.2 \times 10^{-5} \text{ m}^2/\text{s}$
- 4 A furnace emits radiation at 2000 K. Treating it as blackbody radiation, calculate: (a) monochromatic radiation flux at 1 micron wavelength, (b) total emissive power. Given Planck's Law equation,

$$E_{b,\lambda} = \frac{C_1 \lambda^{-5}}{\exp\left(\frac{C_2}{\lambda T}\right) - 1}$$

where  $C_1 = 0.374 \times 10^{-15}$ ,  $C_2 = 1.4388 \times 10^{-2}$

[1+1=2]

- 5 (a) Calculate all the shape factors for the configuration shown below (a sphere of diameter  $d$  inside an equilateral pyramidal box of length  $l = 2d$ ). [3]



- 5 (b) Determine the net radiation heat exchange (per unit area) between two very large parallel plates held at 800 K and 500 K. emissivity is 0.6 for the hot plate and 0.4 for the cold plate. [2]

- 6 Air is flowing over a flat plate 5-m long and 2.5-m wide with a velocity of 2.5 m/s at 15 °C. If  $Pr = 0.7$ , density = 1.20 kg/m<sup>3</sup>, and kinematic viscosity =  $1.5 \times 10^{-5}$  m<sup>2</sup>/s, calculate

- The length of the plate over which the boundary layer is laminar,
- Laminar boundary layer thickness at the point of transition,
- The thickness of the thermal boundary layer at the point of transition,
- Average heat transfer coefficient. [4]

- 7 (a) In a double pipe heat exchanger oil ( $C_p = 2.1$  kJ/kg.°C) at a rate of 90000 kg/h is cooled from 130 °C to 60 °C by water ( $C_p = 4.2$  kJ/kg.°C) that enters at 20 °C and leaves at 45 °C. If the overall heat transfer coefficient is 350 J/m<sup>2</sup>.°C, calculate the heat exchanger area when the exchanger is (i) parallel flow, (ii) counter flow. [4]

- 7 (b) In the counter-flow exchanger design in 7 (a) above, 60000 kg/h oil enters at 130 °C and 75000 kg/h water enters at 20 °C. What will be the outlet temperatures of oil and water achievable in this exchanger? [6]

\*\*\* END OF PAPER \*\*\*

**BITS, PILANI-DUBAI, ACADEMIC CITY, DUBAI**  
**First SEMESTER 2010-2011**  
**CHE C351: Heat Transfer Operations**  
**TEST – II (Open Book)**

DATE: 05.12.2009

DURATION: 50 MINUTES

MAXIMUM MARKS: 15

*Note: Attempt ALL questions. Make suitable design decisions wherever necessary, and mention them clearly. Do not alter any given data.*

**Question 1** [6 Marks]

Air at atmospheric pressure and 20 °C flows past a flat plate at 4 m/s. The plate is heated uniformly throughout its entire length and is maintained at a surface temperature of 60 °C. At 40 cm distance from the leading edge, calculate:

- a) Thickness of the hydrodynamic and thermal boundary layers, using Blasius' exact solution,
- b) Local and average heat transfer coefficients,
- c) Local and average friction coefficients.

*Properties of air at mean temperature are:*

Density = 1.2 kg/m<sup>3</sup>, heat capacity = 1.0 kJ/kg.K,

Thermal conductivity = 0.08 W/m.K, viscosity = 2.2 X 10<sup>-5</sup> kg.m/s

**Question 2** [4 Marks]

A vertical plate 30 cm wide and 1.5 m high is maintained at 70 °C and is exposed to saturated steam at 1 atmospheric pressure. Calculate the heat transfer and the total mass of steam condensed per hour.

*The properties of water film at the mean temperature are:*

Density = 972 kg/m<sup>3</sup>, thermal conductivity = 0.66 W/m.K,

Viscosity = 414 X 10<sup>-6</sup> kg/m.s,  $h_{fg}$  = 2257 kJ/kg.

Assume vapor density is small compared to that of the condensate.

**Question 3** [5 Marks]

3000 kg/h of water is heated from 30 °C to 70 °C by pumping it through a heated pipe of 25 mm diameter. If the surface of the heated pipe is maintained at 110 °C, calculate:

- a) heat transfer coefficient,
- b) Length of the pipe, and
- c) Rate of heat transfer from pipe to water.

*Thermo-physical properties of water at mean bulk temperature (50 °C) are:*

Density = 972 kg/m<sup>3</sup>, viscosity = 0.355 X 10<sup>-3</sup> kg/m.s

Thermal conductivity = 0.7 W/m.°C, heat capacity = 4200 J/kg.°C

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**BITS, PILANI-DUBAI, ACADEMIC CITY, DUBAI**

**First SEMESTER 2010-2011**

**CHE C351: Heat Transfer Operations**

**TEST – I (Closed Book)**

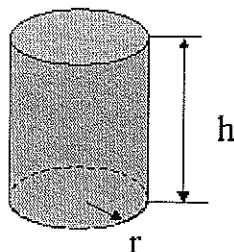
DATE: 24.10.2010

DURATION: 50 MINUTES

MAXIMUM MARKS: 15

**Note: Attempt ALL questions. Mention appropriate units in your answers. Without units, the answer will not be deemed as correct, even if the numerical value is correct.**

1. A thick-walled tube of stainless steel with 2-cm inner diameter and 4-cm outer diameter is covered with a 5-cm layer of insulation ( $k = 0.25 \text{ W/mK}$ ). If the inside wall temperature of the pipe is maintained at  $740^\circ\text{C}$ , calculate the heat loss per meter of length. Also calculate the tube-insulation interface temperature. The temperature at the outside of insulation is  $100^\circ\text{C}$ . Thermal conductivity of steel is  $20 \text{ W/mK}$ . [2+2=4]
2. A very long 4-mm diameter copper rod ( $k = 280 \text{ W/mK}$ ) extends horizontally from a plane heated wall at  $100^\circ\text{C}$ . The temperature of the surrounding air is  $25^\circ\text{C}$  and convective heat transfer coefficient is  $15 \text{ W/m}^2\text{K}$ . Determine the heat loss from the rod. [3]
3. A steel sphere [ $k = 20 \text{ W/m}\cdot^\circ\text{C}$ ] of 4 cm diameter is exposed to a convection environment at  $20^\circ\text{C}$ ,  $h = 15 \text{ W/m}^2\cdot^\circ\text{C}$ . Heat is generated uniformly in the sphere at the rate of  $1.2 \text{ MW/m}^3$ . Calculate the steady state temperature for the center of the sphere? [3]
4. Consider a system of concentric spheres of radius  $r_1$  and  $r_2$  ( $r_2 > r_1$ ). If  $r_1 = 6 \text{ cm}$ , determine the radius  $r_2$  if it is desired to have the value of shape factor  $F_{21}$  equal to 0.6. [2]
5. The radiation shape factor between two flat ends of a hollow cylinder of 10 cm diameter and 10 cm length is 0.17. What is the shape factor of the curved surface of the cylinder with respect to itself? [3]



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**BITS, PILANI-DUBAI, ACADEMIC CITY, DUBAI  
FIRST SEMESTER 2010-2011**

**CHE C351 HEAT TRANSFER OPERATIONS**

**QUIZ - 2 (Closed Book)**

**DATE: 09/12/2010**

**DURATION: 20 MINUTES**

**MAXIMUM MARKS: 7**

Name of the student: -----

I.D.: -----

NOTE: ATTEMPT ALL QUESTIONS. Do rough work on the back of this sheet.

**Question 1.** Consider the following exchanger data: [1 + 1 = 2]

| Flow arrangement    | Fluid | Flow rate, kg/s | Heat capacity, J/kg.°C | Inlet temperature, °C | Outlet temperature, °C |
|---------------------|-------|-----------------|------------------------|-----------------------|------------------------|
| <b>Counter flow</b> | A     | 12.5            | 2530                   | 95                    | 35                     |
|                     | B     | 17.5            | 4180                   | 25                    | ?                      |

Calculate:

Outlet temperature of B =

Heat load of the exchanger =

**Question 2.** Consider the following exchanger data: [2]

| Flow arrangement     | Fluid    | Flow rate, kg/s | Heat capacity, J/kg.°C | Inlet temperature, °C | Outlet temperature, °C |
|----------------------|----------|-----------------|------------------------|-----------------------|------------------------|
| <b>Parallel flow</b> | Methanol | 3.5             | 2200                   | 22                    | 60                     |
|                      | Water    | ?               | 4200                   | 95                    | 65                     |

Overall heat transfer coefficient,  $U = 550 \text{ W/m}^2.\text{K}$

Calculate:

The heat transfer area of the exchanger =

**Question 3.** Consider the following exchanger data: [3]

| Flow arrangement    | Fluid   | Flow rate, kg/s | Heat capacity, J/kg.°C | Inlet temperature, °C | Outlet temperature, °C |
|---------------------|---------|-----------------|------------------------|-----------------------|------------------------|
| <b>Counter flow</b> | Toluene | 4.5             | 2800                   | 98                    | ?                      |
|                     | Water   | 6.5             | 4200                   | 15                    | ?                      |

Overall heat transfer coefficient,  $U = 500 \text{ W/m}^2.\text{K}$

Heat exchanger area =  $90 \text{ m}^2$

Calculate:

Toluene outlet temperature =

BITS, PILANI-DUBAI, ACADEMIC CITY, DUBAI

FIRST SEMESTER 2010-2011

CHE UC351 Heat Transfer Operations

QUIZ – I

(Closed Book)

DATE: 24-11-2010

DURATION: 20 MINUTES

MAXIMUM MARKS: 8

Name of the student: -----

I.D.: -----

**Note: Attempt all questions.**

1. Reynolds analogy is expressed as:

a.  $St = \frac{f}{8}$       b.  $\frac{\tau}{\rho} = \left( \frac{\mu}{\rho} + \varepsilon_M \right) \frac{du}{dy}$       c.  $f = \frac{0.316}{Re_d^{0.25}}$

2. The empirical expression for calculation of heat transfer in fully developed turbulent flow in smooth tubes is:

a.  $Nu_d = 0.0395 Re_d^{3/4}$       b.  $Nu_d = 0.036 Re_d^{0.8} Pr^{1/3} \left( \frac{d}{L} \right)^{0.055}$   
c.  $Nu_d = 0.023 Re_d^{0.8} Pr^n$       d.  $Nu_d = \frac{hd_0}{k} = 4.364$

3.  $\frac{\theta}{\theta_\infty} = \frac{3}{2} \frac{y}{\delta_t} - \frac{1}{2} \left( \frac{y}{\delta_t} \right)^3$  is the equation of thermal boundary layer:

- a. Of laminar flow on a plate
- b. Of turbulent flow on a flat plate
- c. Of laminar flow in a tube
- d. Of turbulent flow in a tube.

4. Which of the following equations is valid for liquid metal heat transfer?

a.  $\frac{\delta}{x} = 0.381 Re_x^{-0.2} - 10256 Re_x^{-1}$       b.  $\frac{\delta}{\delta_t} = 1.64 \sqrt{Pr}$   
c.  $\xi = \frac{\delta_t}{\delta} = \frac{1}{1.026} Pr^{-1/3}$       d.  $\frac{\delta}{x} = \frac{5.0}{Re_x^{0.5}}$

5. The dimensionless number defined as the product of Grashof number and Prandtl number, (Gr.Pr), is called
- a. Peclet number
  - b. Stanton number
  - c. Rayleigh number
  - d. Schmidt number
6. In a blackbody radiation curve, the maximum occurs at  $\lambda_{\max}$ . With increasing temperature,
- a)  $\lambda_{\max}$  will increase
  - b)  $\lambda_{\max}$  will decrease
  - c)  $\lambda_{\max}$  will remain same
  - d)  $\lambda_{\max}$  will increase or decrease depending on the surface properties
7. For a black body
- a) space resistance is equal to zero
  - b) space resistance is equal to unity
  - c) surface resistance is equal to zero
  - d) surface resistance is equal to unity
8. Identify the correct statement regarding radiation shape factors
- a)  $F_{11} = 0$  for plane surface,  $F_{11} > 0$  for convex surface,  $F_{11} < 0$  for concave surface
  - b)  $F_{11} > 0$  for plane surface,  $F_{11} = 0$  for convex surface,  $F_{11} = 0$  for concave surface
  - c)  $F_{11} < 0$  for plane surface,  $F_{11} > 0$  for convex surface,  $F_{11} > 0$  for concave surface
  - d)  $F_{11} = 0$  for plane surface,  $F_{11} = 0$  for convex surface,  $F_{11} > 0$  for concave surface

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